Internal Technical Report

# A Proposal for Efficient Cooling Arrangement for uGMRT Digital Backend 

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Objective : To propose a suitable cooling arrangement for the new Backend system for the uGMRT receiver, so that the system can co-exist with the GMRT Software Backend (GSB) receiver in the Central electronics building (CEB).

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## 1. Introduction :

A Wideband Backend (GWB) is being developed for the uGMRT based on FPGA/GPU hybrid technology. The system being developed utilises Casper designed FPGA / ADC boards to digitise the baseband signals and pass it on to server class machines with GPU cards which work as the compute nodes. The processed data is collected in host nodes through a high speed switch.

As per the current design, Dell make server T620 with two nos. of nVidia GPUs K20 will be used as the compute nodes and three nos. of T620 will be used as host nodes. Final GWB system for 32 antennae will have 5 racks of standard 19 " dimension. Racks 1 to 4 will have compute node machines aongwith Roach units. Each of these racks will consume about 4,850 Watts of maximum power (Typical power consumption 2800 Watts). The fifth rack will have host machines, Switch, small units like PPS, clock and Instruments. This rack will consume about 2050 Watts maximum power (typical consumption 1,250 watts ).

In order to propose the cooling arrangements in the rack, theoritical calculations were done and it was later verified with experiments in the correlator room. A 42 U rack has been used for the experiments, with modifications (as shown in figure) to improve the cool air flow inside the rack. Tests were also done without these modifications for comparison of the cooing effects.

## 2. Summary of experiment results :

A quick summary of the test results is,
a. for a power dissipation of about 4KW per rack ( refer topics"4. GWB modelwise \& 5. GWB rackwise Power Consumption"), the maximum outlet air temperature will be of the order of about 25 degCel., while the maximum inlet temperature is 10 degCel. More details are given in Appendix 3 : Summary of Daily Readings.
b. Volume of cool air from each duct in correlator room as per theoritical assumption is 1000 cfm . It is 1600 cfm as per the measurement done using air flow meter over the duct at duct no. 1 (which is 1200 cfm over duct no. 3). As per the setup 3a (refer apendix II experiment setup details), we gets a temperature difference of 9.21 DegCel . This results in the volume of cool air as 812 cfm . The cool air volume using meter is around $600 \mathrm{cfm}\left(5^{\prime}\right.$ away from the duct).

Test Setup image


Front view


Opening for cool air from the inlet duct

42 U RACK
Total(max.) $\mathrm{WxDxH}=600 \times 1000 \times 2068.5 \mathrm{~mm}$ Usable WxDxH = 500x899x1868.5 mm

Back view


Al. enclosure.


## 3 AC distribution layout :



## MAIN AC PLANT 13,000 CFM

$90 \%$ Effy : 11,700 CFM \& $20 \%$ Leakage : 9,400 CFM

AC Plant : Air Handling Unit (AHU) capacity 13,000 Cubic Feet per Minute (CFM).

## 4 Correlator Room Floor diagram



Total 9*16=144 Blocks.

## 5 GWB Modelwise Power Consumption :

Model wise Power consumption by GWB 30 Ant. System.

| Model Name | Power Consumption/Model |  | Total <br> Models | Total Power Consumptin |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Typical | Maximum |  | Typical | Maximum |
| Dell T620 Nodes | 369 | 628 | 16 | 5,904 | 10,048 |
| Dell T620 Hosts | 369 | 628 | 3 | 1,107 | 1,884 |
| K20 GPU Cards | 105 | 225 | 32 | 3,360 | 7,200 |
| Myricom 10GbE Single Port | 5 | 5 | 32 | 160 | 160 |
| Melox infiniband NIC card | 2 | 2 | 19 | 38 | 38 |
| ROACH Board (1 ADC) | 50 | 60 | 32 | 1,600 | 1,920 |
| ADC in ROACH Boards | 1.5 | 1.5 | 32 | 48 | 48 |
| Melox infiniband 32P Switch | 3 | 3 | 1 | 3 | 3 |
| Signal Generator (N9310A) | 65 | 65 | 1 | 65 | 65 |
| PPS unit \& misc.. | 50 | 100 | 1 | 50 | 100 |

G. Total $\quad 12,335 \quad 21,466$

## 6 GWB Rackwise Power Consumption

Rack wise Power consumption by GWB 30 Ant. System.

| Rack Number1 to 4 | Power Consumption/Model |  |  | Total Models | Total Power Consumptin |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Model Name | Typical | Maximum |  | Typical | Maximum |
|  | Dell T620 Nodes | 369 | 628 | 4 | 1,476 | 2,512 |
|  | K20 GPU Cards | 105 | 225 | 8 | 840 | 1,800 |
|  | Myricom 10GbE Single Port | 5 | 5 | 8 | 40 | 40 |
|  | Melox infiniband NIC card | 2 | 2 | 4 | 8 | 8 |
|  | ROACH Board (1 ADC) | 50 | 60 | 8 | 400 | 480 |
|  | ADC in ROACH Boards | 1.5 | 1.5 | 8 | 12 | 12 |
|  |  |  |  | Sub Total 1 | 2,776 | 4,852 |
| 5 | Dell T620 Hosts | 369 | 628 | 3 | 1,107 | 1,884 |
|  | Melox infiniband NIC card | 2 | 2 | 3 | 6 | 6 |
|  | Melox infiniband 32P Switch | 3 | 3 | 1 | 3 | 3 |
|  | Signal Generator (N9310A) | 65 | 65 | 1 | 65 | 65 |
|  | PPS unit \& misc.. | 50 | 100 | 1 | 50 | 100 |
|  |  |  |  | Sub Total 2 | 1,231 | 2,058 |
|  |  |  |  | G. Total | 12,335 | 21,466 |

## 7 Measuring of air flow using meter :



For air flow measurement in the correlator room, we have used " Pitot Tube Anemometer + Differential Manometer HD350" from Extech Instruments. Measured the air flow volume over the duct number 1 and 3 . Using this meter, we measured air flow at 5 points in the duct ( 4 corners and at the centre about half a foot height) and took average of those readings. It was around $1600 \mathrm{cfm} \& 1200 \mathrm{cfm}$ from duct number $1 \& 3$ respectively.

Detailed measurement of air flow from AC plant to end points in Correlator \& Receiver rooms needs to be done at various possible locations. This will give the complete and exact know how about the cool air volume at different locations and leakages.

The air flow measurement procedure is explained in apendix "VI Air Flow Meter User Manual"

## 8 Measurement of Power Consumption :

Two months back measured the power consumption by the GWB host PC with cards, while correlator code running. The Power consumption was near to Typical Values as per the datasheets/documents. But we need to check again with all the peripherals in the node PC and final code running on it. This may force the power consumption towards maximum ratings.

Setup image and details to be added for future references.

## 9 Theoratical Calculations :

For this theoratical calculations, we have considered the following as inputs :

1. Maximum heat energy from GWB system is 21,500 watts.
2. Cool air entering the correlator room is 6000 CFM (Cubic Feet per Minute).
3. Difference in temperature of air in the corr room $=15$ DegCel from inlet to outlet.
4. Constants :
a. 1 Joule $=0.238902957619$ calories .
b. $1 \mathrm{ft}^{\wedge} 3=0.02832 \mathrm{mtr}^{\wedge} 3$
c. average Air density is $1.128666667 \mathrm{~kg} / \mathrm{mtr}^{\wedge} 3$ for average temperature of 17.5 DegCel.
d. Specific Heat of air @ 20 DegCel \& 1 pascal is 240 calories $/ \mathrm{kg}$ degree celsius.

## A. Maximum Total Heat Energy in calories/min. from the GWB system.

Total Heat Energy in Joules $/$ min. $=$ Total Power consumption * 1 minute
$=21,500$ watts $* 60 \mathrm{sec} .=12,90,000$ Joules $/ \mathrm{min}$.
1 Joule $=0.238902957619$ calories , 1 calorie $=4.186$ joules.
Total Heat Energy in calories $/ \mathrm{min} .=12,90,000 * 0.238902957619=3,08,185$ calories $/ \mathrm{min}$.

## B. Heat Absorption of the cool air in calories/min. from the AHU(AC Plant).

```
Heat Absorption = mass * Specific Heat of air * Diff. in temp.
of the cool air kg/min. Calories/kg degree cel.
    degree celsius
```

a. "mass"
mass $=$ Air Density kg $/ \mathrm{mtr}^{\wedge} 3 *$ Volume $\mathrm{mtr} \wedge 3 / \mathrm{min} .=\mathrm{kg} / \mathrm{min}$.
i. Air Density in $\mathrm{kg} / \mathrm{mtr}^{\wedge} 3$ :
http://www.weatheronline.in/weather/maps/city \& http://www.denysschen.com/catalogue/density.aspx
Air Density @ Pune (above sea level of 1837 feet or 560 mtrs and relative humidity varies between 24 to $94 \%$ in the year 2014) for average relative humidity of $59 \%$ ) is ...

Temperature in DegCel Air Density in $\mathrm{kg} / \mathrm{mtr}^{\wedge} 3$ @ pressure of 1 pascal

10
17
25
1.16
1.13
1.096

So average Air density is $1.128666667 \mathrm{~kg} / \mathrm{mtr}^{\wedge} 3$ for average temperature of 17.5 DegCel.
ii. Air Volume in $\mathrm{mtr}^{\wedge} 3 / \mathrm{min}$ :

$$
1 \mathrm{ft}^{\wedge} 3=0.02832 \mathrm{mtr}^{\wedge} 3 \quad \& 1 \mathrm{mtr}^{\wedge} 3=35.31467 \mathrm{ft}^{\wedge} 3
$$

So volume of cool air reaching corr. Room $=6,000 \mathrm{ft}^{\wedge} 3 / \mathrm{min} .=169.92 \mathrm{mtr}^{\wedge} 3 / \mathrm{min}$.
So mass of cool air $=1.128666667$ * 169.92
reaching corr. Room $\mathrm{kg} / \mathrm{mtr}^{\wedge} 3 \mathrm{mtr}$ ^3/min.
$=191.783040057 \mathrm{~kg} / \mathrm{min}$.
b. "Specific Heat of air $=240$ calories $/ \mathrm{kg}$ degree celsius.
@ temp. of 20degree cel. \&
pressure of 1 pascal
www.usc.edu/org/.../Heat\ Capacity\ and\ Specific\ Heat.pdf , http://www.engineeringtoolbox.com/specific-heat-capacity-d_391.html
c. "Diff. in temperature $=25-10=15$ degree celsius of corr. Room

So,
Heat Absorption $=191.783040057$ * $240 \quad * \quad 15$
of the cool air $\quad \mathrm{kg} / \mathrm{min}$. Calories $/ \mathrm{kg}$ DegCel. per DegCel.
$=6,90,419$ calories $/ \mathrm{min}$.

So,
Heat Absorption of the cool air from each outlet $=6,90,419 / 6$
in the correlator room $=1,15,070 \quad$ Calories $/ \mathrm{min}$.

## Conclusion :

a. Heat absorption of cool air from each inlet of 1000 CFM and inlet to outlet temperature difference of 15 DegCel is $1,15,070$ Calories/minute.
b. So to absorb $3,08,185$ calories $/ \mathrm{min}$. of heat generated by GWB system, we need $2.68(308185 \div 115070)$ cool air inlets. Each inlet of 1000 CFM cool air.

## 10 Practical measurements :

For this practical measurements, we have used the Heat load of 4000 Watts. Maximum GWB power consumption is 21,500 watts or $3,08,185$ calories/minute(refer " 5 . GWB rackwise power consumption").

## Conclusion :

## A. Using 3 fans of 100 cfm each to suck the cool air in the aluminium cage in the rack.

We got the temperature difference of 15.5 degreeCel. So
Volume of cool air per inlet in the correlator room is 482 Cubic Feet/Minute instead of 1000CFM !!!

```
Total Heat Energy
--> 3,08,185 calories/minute
Heat Absorption by 482cfm Cool Air --> 57,312 calories/minute
```

So 5.38 inlets with cool air of 482 CFM are required !!!

## B. Using 2 fans of 600 cfm each to suck the cool air in the aluminium cage in the rack.

If these fans are at $2^{\prime}$ height of the al. cage, then the temperature difference is 5.4 degCel .
If moved these fans at the bottom of the rack and kept this 2 ' height slot in the al. cage open, then the temperature difference is 9.21 degCel. This temperature increases by around 1 degCel, if we put 3 fans of 100 cfm each at the open slot in the al. cage!

If we take 5.4 degCel., then
a. Volume of cool air per inlet in the correlator room is 1384 CFM.
b. Heat Absorption by 1384 CFM Cool Air is $\mathbf{1 , 6 4 , 5 6 5}$ calories/minute.
c. So $\mathbf{1 . 8 7}$ inlets with cool air of $\mathbf{1 3 8 4} \mathbf{C F M}$ are required.

If we take 9.21 degCel., then
a. Volume of cool air per inlet in the correlator room is 812 CFM.
b. Heat Absorption by $\mathbf{8 1 2}$ CFM Cool Air is 96,551 calories/minute.
c. So 3.19 inlets with cool air of $\mathbf{8 1 2} \mathbf{C F M}$ are required.

## 11 Final Conclusions :

A. Theoratically each inlet gives 1000CFM of cool air. So we need 2.68 cool air inlets to absorb the heat generated by GWB system of 21,500 Watts ( $3,08,185$ calories/minute). But practically, with 3 fans of 100 cfm each to suck the cool air into the al. cage, we need 5.38 cool air inlets. This is doubble the theoratical requirement. So we have to find out the reasons, which could be as follows ;

1. Actual cool air spilt for the correlator room may not be 6000 CFM.
2. There may be leakeages between the AHU and correlator room inlets.
3. May be considerable leakages at the bottom of the rack.
4. May be adjustment of locations of the heat load is required in the rack, in order to utilize the cool air efficiently.
5. May be adjustment of locations of the input/output thermometers is required to findout the actual volume of the cool air, which may be more than 482 CFM. 6. If possible and not harmful to the system, increase the humidity of the cool air (decrease the temperature of the cool air comming out of the inlets).
B. We found the following things are useful in utilising the cool air efficiently.
6. We have replaced the 3 fans of 100 cfm each to suck the cool air into the al. cage by 2 fans of 600 cfm at 2 ' height in the al. cage.
7. Made sure the cool air entered in the rack, shouldn't leakout.
8. And also took care of no obstacles in the cool air flow towards the exit in the rack.

Enenthough the rack is kept 5 feet away from the cool air duct and without any aluminium cage/duct to bring the cool air from the inlet(setup No. 3a), we got a averaged temperature difference of 5.4 degCel.

If we move these fans from the al. cage to the bottom of the rack and keep the slot in the al. cage @ $2^{\prime}$ height open, then the temperature difference increses to 9.21 degCel.

This temperature increases by around 1 degCel, if we put 3 fans of 100 cfm each at the open slot in the al. cage!

If we take maximum of these ie 9.21 DegCel , then the volume of cool air from the outlet is 812 cfm .

So we need 3.19 inlets of 1000 cfm cool air to dissipate $21,500 \mathrm{KW}$ heat from GWB system.

## Appendices will follow in the next page

## Appendices :

## I Auto calculator :

please use the file auto_calc_temp.ods to change the input parameters to get output values. Sample file attached here for standard input parameters...

## User Parameters :

PowerConsumption in Watts -->
Difference in Temperature -->
Volume of Cool Air in ft $\mathrm{ft}^{\wedge}$-->
Heat absorption in watts -->
Difference in temperature deg. Cel. -->
Constant Parameters :
1 Joule equal to --> Calories
1 Cubic feet equal to --> Cubic mtr
Air Density kg/mtr 3 @ 25 dC \& 1 Pascal
Specific Heat of Air cal/kgdC @ 20dC \& 1 Pascal

## INPUTS

```
2 1 5 0 015.5
```812

4,000
9.21
0.2389029576
0.02832 1.128666667

240

Tables:
Total Heat Energy in Cal./min.
For 1000 watts of Power Consumption -->
For 1500 watts of Power Consumption -->
For 2000 watts of Power Consumption -->
For 2500 watts of Power Consumption -->
For 3000 watts of Power Consumption -->
For 3500 watts of Power Consumption -->
For 4000 watts of Power Consumption -->
For 4500 watts of Power Consumption -->
For 5000 watts of Power Consumption -->

Constant Values Intermittent Calculation Parameters: int'nt. results

\section*{Output Parameters}

\section*{Total Heat Energy in calories/minute --> \\ Heat Absorption of Cool Air cal/min --> \\ Number of cool air inlets required..}

Volume of Cool Air in \(\mathrm{ft}^{\wedge} 3\) :

Volume of Cool Air in mtr^3 --> 22.99584
Mass of cool air in kg/minute --> 25.954638088

Volume of Cool Air in \(\mathrm{ft}^{\wedge} \mathbf{3}\) @1KW heat if :
Temperature difference of 3 deg. Cel.
Temperature difference of 3.5 deg . Cel.
2,135
28,668 Temperature difference of 4 deg. Cel. 1,869
35,835 Temperature difference of 4.5 deg. Cel. 1,661
43,003
50,170
57,337
64,504
71,671
Temperature difference of 5 deg. Cel. 1,495
Heat Absorption of Cool Air in cal./min. For 1000ft^3/
For Temperature difference of 1 deg.Cel. 6,229

OUTPUTS
3,08,185
96,551
3.19

For Temperature difference of 1.5 deg.Cel. 9,344
For Temperature difference of 2 deg.Cel. 12,458
For Temperature difference of 2.5 deg.Cel. 15,573
For Temperature difference of 3 deg.Cel. 18,687
For Temperature difference of 3.5 deg.Cel. 21,802
For Temperature difference of 4 deg.Cel. 24,916
For Temperature difference of 4.5 deg.Cel. 28,031
For Temperature difference of 5 deg.Cel. 31,146
For Temperature difference of 5.5 deg.Cel. 34,260
For Temperature difference of 6 deg.Cel. 37,375
For Temperature difference of 6.5 deg.Cel. 40,489
For Temperature difference of 7 deg.Cel. 43,604
For Temperature difference of 7.5 deg.Cel. 46,718
For Temperature difference of 8 deg.Cel. 49,833
For Temperature difference of 8.5 deg.Cel. 52,947
For Temperature difference of 9 deg.Cel. 56,062
For Temperature difference of 9.5 deg.Cel. 59,177
For Temperature difference of 10 deg.Cel. 62,291

\section*{II Experiment setup details :}

The following setups were done with the help of mechanical and electrical sections. These setups were used to study the effects of cooling over varying temperature under different conditions like when the rack is placed over the inlet cool air duct or 5 feet away from it and so on.... A total of 8 setups were created to study the possible conditions to make out the rise in temperature at the output over input ie difference in temperature of the air entering the rack and exiting the rack.

Setup 1 : 42U Rack directly on the cool air inlet in the corr room with all the leakages at the bottom closed. The cool air enters the rack from the front bottom opening and enters the aluminium cage through a window provided in the cage at a height of about 2 feet. One thermometer(T1) and two temperature sensors (S2 \& S3) are present at this opening to measure the inlet temperature. Inside the al. cage about a feet high 4 hellogen (with seperate ON-OFF switches in the two extension boards having current carrying capacity of 16 amps. These two extension boards are getting power from MSEB metal MCB box through 25amps MCB) lamps of 1 KW each are mounted. Some 3 feet above these lamps there is window in the al cage at the backside for heated air to exhaust. At this poist also one thermometer (T2) and two sensors (S4 \& S5) are mounted. Sensor S1 is outside and near to this rack.

Rabbit card gets input on pin numbers 1 to 5 on 64pin FRC male connector from sensors S1 to S5 respectively. +11 volts power has been given to this rabbit card as well all to all temperature sensor cards. An ethernet cable with ip 192.168.4.152 has been connected to this rabbit card for web based monitoring of temperature.

Setup 2 : setup 1 with 3 fans of 100 cfm each, mounted at the height of about 2 feet front side, to suck air in the aluminium cage.

Setup 2a : setup 2 with 2 fans of 600cfm each, mounted at the height of about 2 feet front side, to suck air in the aluminium cage. This is done with and without aluminium plate in al. cage to abstruct the air.

Setup 3 : setup \(1 \& 2\) with whole setup moved about 5 feet away from cool air inlet duct. Cool air brought to the rack's inlet using aluminium sheet enclosure underneath the false flooring.

Setup 3a : setup 3 with 2 fans of 600 cfm each instead 3 fans with 100 cfm each and without al. enclosure to bring the cool air to rack. This is done with and without an aluminium plate in the al. cage in the rack, to be an obstacle in the air flow.

Setup 3b : setup 3a with 2 fans of 600cfm each, moved to bottom of the rack with opening @ 2' height. This is done with and without an aluminium plate in the al. cage in the rack, to be an obstacle in the air flow.

Setup 4 : setup 3 without fans to suck the cool air in the aluminium cage at 2 feet height.
Setup 5 : setup 3 without aluminium sheet enclosure underneath the false flooring to bring the cool air from the inlet duct to the rack and with fans to suck the air in the aluminium cage at 2 feet height.

Setup 6 : setup 5 but without fans to suck the cool air in the aluminium cage at 2 feet height.
Setup 7 : Rack without any modifications (aluminium cage in the rack removed). Kept 5 feet away from the cool air inlet duct. Input temperature monitoring about 2 feet high at the frontside. Four
hellogens of 1 KW each mounted at the centre of the rack. And output temperature monitoring at the top of the rack's rear side.

Setup 8 : Rack without any modifications (aluminium cage in the rack removed). Kept over the cool air inlet duct. Input temperature monitoring about 2 feet high at the frontside, where cool air blows in the rack's bottom. Four hellogens of 1 KW each mounted at the centre of the rack. And output temperature monitoring at the top of the rack's rear side.

T - Thermometer readings average. \& S - Temperature Sensors readings average in Deg. Cel. LH - Left Hand side RH - Right Hand side (from back) F- Front of rack B - Back of rack.

\section*{III Summary of Daily readings :}
\begin{tabular}{|c|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Setup No.} & \multirow[t]{2}{*}{Measuring Instrument} & \multicolumn{4}{|l|}{Average temp. readings for} & \multirow[t]{2}{*}{Max. output temp. for 4KW} & \multirow[t]{2}{*}{Termp. Sensor minus Thermometer readings.} \\
\hline & & 1KW & 2KW & 3KW & 4KW & & \\
\hline \multirow[t]{2}{*}{I} & Themometer & 3.0 & 6.6 & 11.2 & -- & \multirow[t]{2}{*}{20.0} & \multirow[t]{2}{*}{1.2} \\
\hline & Temperature Sensor & 4.2 & 8.2 & 12.2 & -- & & \\
\hline \multirow[t]{2}{*}{II} & Themometer & 2.6 & 6.3 & 9.0 & 11.7 & \multirow[t]{2}{*}{22.5} & \multirow[t]{2}{*}{2.2} \\
\hline & Temperature Sensor & 4.3 & 8.2 & 11.7 & 14.2 & & \\
\hline \multirow[t]{2}{*}{IIa} & Themometer/TempSe & 6.23 & \multicolumn{3}{|l|}{\multirow[t]{2}{*}{\begin{tabular}{l}
8.89 with al. plate \\
\(6.06 \mathrm{w} / \mathrm{o}\) al. plate
\end{tabular}}} & \multirow[t]{2}{*}{\[
\begin{aligned}
& 16.00 \\
& 17.00
\end{aligned}
\]} & \multirow[t]{2}{*}{\[
\begin{aligned}
& 2.26 \\
& 2.56
\end{aligned}
\]} \\
\hline & Themometer/TempSe & 3.50 & & & & & \\
\hline \multirow[t]{2}{*}{III} & Themometer & 2.8 & 7.3 & 12.5 & 14.8 & \multirow[t]{2}{*}{25.5} & \multirow[t]{2}{*}{2.6} \\
\hline & Temperature Sensor & 5.3 & 10.0 & 14.9 & 17.6 & & \\
\hline \multirow[t]{2}{*}{IIIa} & Themometer/TempSe & 9.21 & \multicolumn{3}{|l|}{\multirow[t]{2}{*}{\begin{tabular}{l}
9.23 with al. plate \\
5.33 w/o al. plate
\end{tabular}}} & 20.00 & 0.02 \\
\hline & Themometer/TempSe & 6.12 & & & & 21.00 & -0.79 \\
\hline \multirow[t]{2}{*}{IIIb} & Themometer/TempSe & 10.64 & \multicolumn{3}{|l|}{\multirow[t]{2}{*}{13.26 with al. plate 12.84 w/o al. plate}} & 20.50 & 2.62 \\
\hline & Themometer/TempSe & 10.82 & & & & 20.00 & 2.02 \\
\hline \multirow[t]{2}{*}{IV} & Themometer & 3.5 & 10.0 & 14.1 & 18.5 & \multirow[t]{2}{*}{28} & \multirow[t]{2}{*}{2.8} \\
\hline & Temperature Sensor & 6.4 & 12.6 & 17.1 & 21.4 & & \\
\hline \multirow[t]{2}{*}{V} & Themometer & 4.8 & 9.5 & 11 & 18.4 & \multirow[t]{2}{*}{33} & \multirow[t]{2}{*}{-3.0} \\
\hline & Temperature Sensor & 3.8 & 6.5 & 9.3 & 12.1 & & \\
\hline \multirow[t]{2}{*}{VI} & Themometer & 8.0 & 18.2 & 24.8 & 31.7 & \multirow[t]{2}{*}{46} & \multirow[t]{2}{*}{1.8} \\
\hline & Temperature Sensor & 10.5 & 20.0 & 26.8 & 32.7 & & \\
\hline \multirow[t]{2}{*}{VII} & Themometer & -5.2 & -3.2 & -1.3 & -1.1 & \multirow[t]{2}{*}{31.5} & \multirow[t]{2}{*}{1.9} \\
\hline & Temperature Sensor & -0.8 & -1.3 & -2.3 & +1.2 & & \\
\hline \multirow[t]{2}{*}{VIII} & Themometer & -2.2 & -3.1 & -3.1 & -2.8 & \multirow[t]{2}{*}{9.0} & \multirow[t]{2}{*}{1.0} \\
\hline & Temperature Sensor & -0.5 & -1.4 & -2.0 & -3.3 & & \\
\hline
\end{tabular}

Note : Thermometer readings are considered.

My words on this summary :
1. Since maximum temperature crosses 26 degCel ( 16 degCel over the temperature at the duct) for the setups IV and above, not considered for further evaluation.
2. The cool air entering the rack must be sucked towards the loads and heat air must be sucked out.
3. Temperature goes down by about 0.5 to 2 degCel for 1 KW to 4 KW load, if we use fans to suck the air when the rack is over the duct. But this trend reverses when the rack is 5 feet away from the duct with fans.
4. Temperature for 1 KW about 3 degCel. It increases by about 3.5 degCel from 1 KW to 2 KW . And by 3 to 5 degCel \& about 2.5 degCel for 2 to 3 KW and 3 to 4 KW respectively.
5. Setup 2a, 3a \& 3b done after all other setups, to make use of the cool air efficiently. We got succeded in using it almost \(100 \%\) efficiently by using 2 fans of 600 cfm each to suck the cool air into the rack. Set up 3a \& 3b with some modifications may be used for final setup.

\section*{IV Average of Daily readings :}

\section*{SETUP: 1}

42U Rack directly placed on the cool air inlet in the correlator room with all the leakages at the bottom closed. Aluminium cage with cool air in and hot air out and load in the centre of the cage.


SETUP: 2
setup 1 with 3 fans of 100 cfm mounted at the height of about 2 feet front side, to suck air in the aluminium cage.
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{2}{|l|}{Difference in Temperature} & \multirow[b]{2}{*}{Heat load of KW's} & \multirow[b]{2}{*}{Time} & & \multirow[b]{2}{*}{Remarks} \\
\hline & T & \(\underline{\text { S }}\) & & & & \\
\hline \multirow[t]{4}{*}{28/10/2014} & 2 & 4.2 & 1 & 10:00-11:40 & LHB & every 10 min . \\
\hline & 8.5 & 9 & 2 & 11:45-13:00 & LHB + RHB & \\
\hline & 10 & 12.5 & 3 & 14:10-15:30 & LH-2 + RH & " \\
\hline & 13 & 15.5 & 4 & 15:40-16:40 & All & " \\
\hline \multirow[t]{4}{*}{29/10/2014} & 11 & 13.5 & 4 & 09:40-11:30 & All & every 10 min . \\
\hline & 10 & 12.5 & 3 & 11:40-13:00 & LH-2 + RHF & " \\
\hline & 7.5 & 9.5 & 2 & 14:20-15:20 & LHB + RHB & " \\
\hline & 2.5 & 4.5 & 1 & 15:55-17:10 & LHB & " \\
\hline \multirow[t]{6}{*}{30/10/2014} & 2 & 4.2 & 1 & 09:45-10:40 & LHF & every 10 min . \\
\hline & 2 & 4 & 1 & 10:50-11:45 & RHF & " \\
\hline & 4 & 5.2 & 1 & 12:00-12:55 & RHB & " \\
\hline & 2.5 & 4 & 1 & 14:10-15:00 & LHB & " \\
\hline & 5 & 7 & 2 & 15:30-16:10 & LH-2 & " \\
\hline & 5.5 & 8 & 2 & 16:20-17:20 & RH-2 & " \\
\hline \multirow[t]{3}{*}{31/10/2014} & 6.2 & 7.7 & 2 & 09:50-11:30 & LHF+RHB & every 10 min . \\
\hline & 5.2 & 7.7 & 2 & 11:35-14:20 & LHB+RHF & " \\
\hline & 7.2 & 10.3 & 3 & 14:25-16:40 & LH-2+RHF & " \\
\hline 31/10/2014 & 11 & 13.5 & 4 & 09:45-16:00 & All & every 10 min . \\
\hline 26/05/2015 & 8.0 & 10.5 & 4 & 14:30-16:50 & All & every 10 min . \\
\hline Page 19 & & & & w/o al. plate o & bbstacke in the & al. cage. \\
\hline
\end{tabular}

Average Readings in degCel.
1KW 2KW 3KW 4KW
\begin{tabular}{lccccc} 
Themometer & 2.6 & 6.3 & 9.0 & 11.7
\end{tabular}

Conclusions: Temperature Sensor results are about 2 degCel. more than the Themometer readings. And maximum output temperature for 4 KW is about 22.5 (1/11/2014@14:50hrs) Alos noted a decrease of temperature by \(\mathbf{3}\) degCel if remove the al. plate in the al. cage.

SETUP: 2a
setup 2 with \(\underline{2}\) fans of 600 cfm mounted at the height of about 2 feet front side, to suck air in the aluminium cage.
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{3}{|l|}{Difference in Temperature} & \multicolumn{2}{|l|}{\multirow[b]{2}{*}{Time}} & \multirow[b]{2}{*}{Remarks} \\
\hline & T & \(\underline{S}\) & KW's & & & \\
\hline 03/06/2015 & 6.23 & 8.89 & 4 & 12:40-17:00 & & every 10 min . \\
\hline Page 20 & & & & with al. plate & obsta & al. cage. \\
\hline 05/06/2015 & 3.50 & 6.06 & 4 & 10:20-17:00 & All & every 10 min . \\
\hline Page 20 & & & & w/o al. plate & bsta & al. cage. \\
\hline
\end{tabular}

SETUP: 3
setup \(1 \& 2\) with whole setup moved about 5 feet away from cool air inlet duct. Cool air brought to the rack's inlet using aluminium sheet enclosure underneath the false flooring.

Difference in
Temperature
Date Readings Heat load of

07/11/2014
\begin{tabular}{lll}
\(\underline{\mathbf{T}}\) & \(\underline{\mathbf{S}}\) & \(\underline{\text { KW's }}\) \\
\hline .3
\end{tabular}

Time \(\overline{\text { 15:20-17:10 RHB+RHF }}\)

Remarks every 10 min .
Sheet 1 10/11/2014 sheet 2
11/11/2014
sheet 3
12/11/2014
sheet 4
13/11/2014
sheet 5
14/11/2014
sheet 6
15/11/2014
sheet 7

16/11/2014
sheet 8
16.0 17.82 4 10:00-13:00 All "
\(\begin{array}{llll}12.72 & 15.25 & 3 & 14: 15-17: 10 \\ \text { LH-2+RHB " }\end{array}\)
\(\begin{array}{llll}2.28 & 5.12 & 1 & 10: 00-14: 20 \\ \text { LHF }\end{array}\)
\(\begin{array}{llll}6.50 & 8.74 & 2 & 14: 50-17: 15 \\ \text { RH-2 " }\end{array}\)
\(6.19 \quad 9.36\)
\(7.00 \quad 10.08\)
\(7.50 \quad 10.54\)
1.934 .04
\(3.93 \quad 6.13\)
\(2.81 \quad 5.53\)
\(3.30 \quad 5.95\)
\(9.64 \quad 12.02\)
12.3314 .63
13.7016 .38
14.7918 .67

10:00-14:20 LH -2
14:10-15:15 LHB+RHF
09:50-14:35 LHF+RHB
14:40-17:10 RHF
10:20-14:40 RHF
14:45-17:20 LHB
10:10-11:30 RHB
11:35-14:20 LHB+RHB
14:30-15:50 LHB+RH-2
15:55-17:20 All
09:25-20:10 All

Average Readings in degCel.
1KW 2KW 3KW 4KW
Themometer \(\begin{array}{llll}2.8 & 7.3 & 12.5 & 14.8\end{array}\)
\(\begin{array}{lllll}\text { Temp. Sensors } & 5.3 & 10.0 & 14.9 & 17.6\end{array}\)
Conclusions : Temperature Sensor results are about 2.5 degCel. more than the Themometer readings. And maximum output temperature for 4 KW is about 25.5 (16/11/2014@12:00hrs sheet no. 8 of temp_readings.ods file.)

Setup 3a : setup 3 with 2 fans of 600 cfm each instead 3 fans with 100 cfm each.
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{3}{|l|}{Difference in Temperature} & \multicolumn{2}{|l|}{\multirow[b]{2}{*}{Time \(\quad\) Remarks}} \\
\hline & T & \(\underline{S}\) & KW's & & \\
\hline 08/06/2015 & 6.12 & 5.33 & 4 All & 15:10-17:10 & every 10 min . \\
\hline Page 21 & & & & w/o al. plate o & al. cage. \\
\hline 09/06/2015 & 9.21 & 9.23 & 4 All & 10:00-17:10 & every 10 min . \\
\hline Page 21 & & & & with al & e in the al. cage. \\
\hline 16/06/2015 & 9.89 & 12.74 & 4 All & 14:40-17:10 & late " \\
\hline
\end{tabular}

Setup 3b : setup 3a with 2 fans of 600 cfm each moved to bottom of the rack with opening @ \(2^{\prime}\) height.
\begin{tabular}{lrlccc}
\begin{tabular}{l}
\(17 / 06 / 2015\)
\end{tabular} & 10.64 & 13.26 & 4 All \(10: 10-17: 10\) & with al. plate obstacke in the al. cage. \\
\begin{tabular}{l} 
Page 22
\end{tabular} & & & & every 10 min.
\end{tabular}

\section*{SETUP : 4}
setup 3 but without fans to suck the cool air in the aluminium cage at \(\mathbf{2}\) feet height.
\{Sheet number 13 \& 14 in the temp_readings.ods file\}
Difference in
Temperature
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{2}{|l|}{Readings} & \multirow[t]{2}{*}{Heat load of KW's} & \multirow[b]{2}{*}{Time} & & \multirow[b]{2}{*}{Remarks} \\
\hline & T & \(\underline{S}\) & & & & \\
\hline 26/12/2014 & 18.00 & 21.30 & 4 & 11:50-15:30 & & every 10 min . \\
\hline Sheet 13 & 14.80 & 17.69 & 3 & 15:40-17:10 & LH-2+RHF & / \\
\hline 27/12/2014 & 11.54 & 13.33 & 2 & 09:55-12:00 & LHB+RHB & every 10 min . \\
\hline Sheet 13 & 04.53 & 06.61 & 1 & 12:10-16:10 & LHB & " \\
\hline & 18.33 & 21.15 & 4 & 16:20-21:00 & All & \\
\hline 28/12/2014 & 13.44 & 16.48 & 3 & 12:20-15:00 & LHF+RH-2 & every 10 min . \\
\hline Sheet 14 & 08.57 & 11.99 & 2 & 15:10-16:30 & LHF+RHF & " \\
\hline & 02.50 & 06.15 & 1 & 16:40-20:30 & RHF & " \\
\hline 29/12/2014 & 19.21 & 21.70 & 4 & 10.10-16:40 & All & every 10 min . \\
\hline
\end{tabular}

Sheet 14

Note : In this setup when bottom of the rack wasn't packed properly, so cool air was passing there. The average readings were about 1 to 4 degrees higher than the above mentioned ( Readings in the Sheet number 9 of temp_readings.ods file) readings for 1 KW to 4 KW load resp.. Above readings are taken again after packing the rack bottom with the floor.

Average Readings in degCel.
1KW 2KW 3KW 4KW
\(\begin{array}{llllll}\text { Themometer } & 3.5 & 10.0 & 14.1 & 18.5\end{array}\)
\(\begin{array}{llllll}\text { Temp. Sensors } & 6.4 & 12.6 & 17.1 & 21.4\end{array}\)
Conclusions : Temperature Sensor results are about 1.2degCel. more than the Themometer readings. And maximum output temperature for 4 KW is about 28.0 (26,27,29/11/2014 sheets 13 \& 14)

SETUP: 5
setup moved 5 feet away from the inlet cool air duct, without aluminium duct between cool air inlet and rack but fans to suck air at the aluminium cage present.
\{Sheet number 11 in the temp_readings.ods file\}
Difference in
Temperature
Date

26/11/2014
Sheet 11
27/11/2014
Sheet 11

28/11/2014
Sheet 11
31/12/2014
Readings Heat load of
\(\underline{\mathrm{T}} \quad \underline{\mathrm{S}} \quad \underline{\text { KW's }}\) \(\qquad\)
Time
Remarks
sheet 16
01/01/2015
sheet 16
02/01/2015
sheet 16
03/01/2015
\begin{tabular}{ll}
11.90 & 08.23 \\
09.60 & 06.86 \\
17.95 & 11.33 \\
06.45 & 05.35 \\
09.20 & 06.14 \\
18.31 & 11.37
\end{tabular}

3
09:50-11:30 LHF
every 10 min .
2 12:00-13:00 LHF+RHF
every 10 min .
\(09.20 \quad 06.14\)
4 11:30-13:00 All
1 14:20-16:00 LHB
"
every 10 min.
\(18.73 \quad 12.17\)
\(4 \quad\) 10:30-16:30 All
66
3
\begin{tabular}{ll} 
10:00 - 11:30 LH-2+RHB & \("\) \\
14.15 - 16:00 LHF+RHB & \("\) \\
10:10 - 14:10 RHF & \("\) \\
15:00 - 16:00 All & " \\
10:00 - 12:10 All & \("\)
\end{tabular}

Note : In this setup Thermometer reading are higher than the sensor readings!

\section*{Themometer}

Average Readings in degCel. 1KW 2KW 3KW 4KW
\begin{tabular}{llllll} 
Temp. Sensors & 3.8 & 6.5 & 9.3 & 12.1
\end{tabular}

Max. o/p Themometer +/-
temp. 4KW Temp. Sensor

Conclusions : Temperature Sensor results are about 1 to 6 degCel. less than the Themometer readings. And maximum output temperature for 4 KW is about 33degCel (sheets 11 and 16)

SETUP: 6
setup 5 but without fans to suck the cool air in the aluminium cage at \(\mathbf{2}\) feet height. \{Sheet number 12 in the temp_readings.ods file\}
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{2}{|l|}{Difference in Temperature Readings} & \multirow[b]{2}{*}{Heat load of KW's} & \multirow[b]{2}{*}{Time} & \multirow[b]{2}{*}{Remarks} \\
\hline & T & \(\underline{S}\) & & & \\
\hline 28/11/2014 & 08.00 & 10.55 & 1 & 14:30-16:00 LHF & every 10 min . \\
\hline Sheet 12 & 18.70 & 20.61 & 2 & 16:20-17:00 LHF+RHB & " \\
\hline 01/12/2014 & 23.85 & 26.10 & 3 & 10:00-12:00 LHF+RH-2 & every 10 min . \\
\hline Sheet 12 & 18.18 & 19.96 & 2 & 12:10-14:50 LHB+RHF & \\
\hline 02/12/2014 & 32.36 & 34.08 & 4 & 10:00-12:50 All & every 10 min . \\
\hline Sheet 12 & 17.67 & 19.64 & 2 & 14:10-15:30 LH-2 & " \\
\hline 30/12/2014 & 30.11 & 30.20 & 4 & 12:20-15:50 All & every 10 min . \\
\hline 03/01/2015 & 32.56 & 33.87 & 4 & 12:30-17:00 All & \\
\hline \multirow[t]{3}{*}{\begin{tabular}{l}
\[
05 / 01 / 2015
\] \\
sheet 15
\end{tabular}} & 25.84 & 27.50 & 3 & 12:10-17:10 LHF+RF-2 & " \\
\hline & \multicolumn{4}{|l|}{Average Readings in degCel.} & \\
\hline & 1KW & 2KW & 3KW 4KW & & \\
\hline Themometer & 8.0 & 18.2 & 24.831 .7 & & \\
\hline Temp. Sensors & 10.5 & 20.0 & \(26.8 \quad 32.7\) & & \\
\hline \multicolumn{6}{|l|}{Conclusions : Temperature Sensor results are about 1 to 2 degCel. more than the Themometer readings. And maximum output temperature for 4 KW is about 46 degCel. (sheet no. 12)} \\
\hline
\end{tabular}

\section*{SETUP: 7}

Rack without any modifications (aluminium cage in the rack removed). Kept 5 feet away from the cool air inlet duct. \{Sheet number 17 in the temp_readings.ods file\}
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{2}{|l|}{Readings} & \multirow[t]{2}{*}{Heat load of KW's} & \multirow[b]{2}{*}{Time} & & \multirow[b]{2}{*}{Remarks} \\
\hline & & \(\underline{\text { S }}\) & & & & \\
\hline 22/01/2015 & -1.14 & 1.24 & 4 & 11:40-16:00 & All & every 10 min . \\
\hline \multicolumn{7}{|l|}{Sheet 17} \\
\hline 23/01/2015 & -1.26 & 2.27 & 3 & 10:00-14:20 & LHB+RH-2 & every 10 min . \\
\hline Sheet 17 & -3.22 & 1.27 & 2 & 14:30-15:50 & LHB+RHB & " \\
\hline & -5.25 & -0.79 & 1 & 16:00-17:20 & RHB & " \\
\hline
\end{tabular}

Average Readings in degCel.
1KW 2KW 3KW 4KW
\(\begin{array}{lllll}\text { Themometer } & -5.2 & -3.2 & -1.3 & -1.1\end{array}\)
Temp. Sensors \(\quad-0.8 \quad-1.3 \quad-2.3 \quad+1.2\)
Conclusions : Temperature Sensor results are about 1 to 4 degCel. less than the Themometer readings. And maximum output temperature for 4 KW is about 31.5 on 22/01/2015 sheet no. 17 .

SETUP : 8
Rack without any modifications (aluminium cage in the rack removed). Kept over the cool air inlet duct. \{Sheet number 17 in the temp_readings.ods file\}

Difference in
Temperature
\begin{tabular}{|c|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{Date} & \multicolumn{2}{|l|}{Readings} & \multirow[t]{2}{*}{Heat load of KW's} & \multirow[b]{2}{*}{Time} & & \multirow[b]{2}{*}{Remarks} \\
\hline & T & \(\underline{S}\) & & & & \\
\hline 24/01/2015 & -2.78 & -3.35 & 4 & 12:30-17:10 & All & every 10 min . \\
\hline \multicolumn{7}{|l|}{Sheet 18} \\
\hline 30/01/2015 & -3.17 & -2.20 & 3 & 10:30-14:50 & LHB+RH-2 & " \\
\hline Sheet 18 & -3.21 & -1.70 & 2 & 15:00-17:10 & LHB+RHB & " \\
\hline 02/02/2015 & -2.21 & -0.50 & 1 & 10:00-12:00 & LHF & " \\
\hline Sheet 18 & -3.04 & -1.10 & 2 & 12:10-15:00 & LH-2 & " \\
\hline & -3.14 & -1.68 & 3 & 15:10-17:20 & LH-2+RHB & " \\
\hline
\end{tabular}

Average Readings in degCel.
1KW 2KW 3KW 4KW
\(\begin{array}{lllll}\text { Themometer } & -2.2 & -3.1 & -3.1 & -2.8\end{array}\)
\(\begin{array}{lllll}\text { Temp. Sensors } & -0.5 & -1.4 & -2.0 & -3.3\end{array}\)
Conclusions : Temperature Sensor results are about 1.5 degCel. more than the Themometer readings. And maximum output temperature for 4 KW is about 9.0 degCel 24/01/2015 sheet no. 18.

\section*{Conclusions :}
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multirow[b]{2}{*}{Setup No.} & \multicolumn{4}{|l|}{Temperature Difference Range, for the Heat Load of} & \multirow[b]{2}{*}{Remarks} \\
\hline & 1 KW & \(\underline{2} \mathrm{KW}\) & 3 KW & 4 KW & \\
\hline 1 & \[
\begin{aligned}
& 2-4 \\
& 4-4.5
\end{aligned}
\] & \[
\begin{aligned}
& 5.5-8 \\
& 8-8.5
\end{aligned}
\] & \[
\begin{aligned}
& 10.5-12 \\
& 11.5-13
\end{aligned}
\] & not done not done & Thermometer Temp. Sensors \\
\hline Average & 3.2 & 7 & 11.7 & not done & \\
\hline 2 & \[
\begin{aligned}
& 2-4 \\
& 4-5.2
\end{aligned}
\] & \[
\begin{aligned}
& 5-8.5 \\
& 7-9.5
\end{aligned}
\] & \[
\begin{aligned}
& 7.2-10 \\
& 10.3-12.5
\end{aligned}
\] & \[
\begin{aligned}
& 11-13 \\
& 13.5-15.5
\end{aligned}
\] & Thermometer Temp. Sensors \\
\hline Average & 3.6 & 7.2 & 9.8 & 13.2 & \\
\hline 3 & \[
\begin{aligned}
& 2-4 \\
& 4-6
\end{aligned}
\] & \[
\begin{aligned}
& 6-9.5 \\
& 8.7-12
\end{aligned}
\] & \[
\begin{aligned}
& 12.3-12.7 \\
& 14.6-15.3
\end{aligned}
\] & \[
\begin{aligned}
& 13.7-16.0 \\
& 16.4-18.7
\end{aligned}
\] & Thermometer Temp. Sensors \\
\hline Average & 4 & 8 & 13.8 & 16.2 & \\
\hline 4 & \[
\begin{aligned}
& 2.5-4.5 \\
& 6.2-6.6
\end{aligned}
\] & \[
\begin{aligned}
& 8.5-11.5 \\
& 12.0-13.3
\end{aligned}
\] & \[
\begin{aligned}
& 13.5-14.8 \\
& 16.5-17.7
\end{aligned}
\] & \[
\begin{aligned}
& 18.0-19.2 \\
& 21.2-21.7
\end{aligned}
\] & Thermometer Temp. Sensors \\
\hline Average & 4.6 & 10.9 & 15.6 & 19.8 & \\
\hline
\end{tabular}
\begin{tabular}{|c|c|c|c|c|c|}
\hline \multirow[t]{2}{*}{5} & 3.2-6.4 & 9.2-9.6 & 10.1-11.9 & 17.8-19.1 & \multirow[t]{2}{*}{Thermometer Temp. Sensors} \\
\hline & 2.2-5.3 & 6.1-6.8 & 8.2-10.5 & 11.4-13.1 & \\
\hline Average & 4.3 & 7.8 & 10.0 & 10.4 & \\
\hline \multirow[t]{2}{*}{6} & 8.0 & \(17.7-18.7\) & 23.8-25.9 & 30.1-32.5 & \multirow[t]{2}{*}{Thermometer Temp. Sensors} \\
\hline & 10.5 & 19.6-20.6 & 26.1-27.5 & 30.2-34.0 & \\
\hline Average & 9.2 & 19.2 & 25.8 & 32.0 & \\
\hline \multirow[t]{2}{*}{7} & -5.25 & -3.22 & -1.26 & -1.14 & \multirow[t]{2}{*}{Thermometer Temp. Sensors} \\
\hline & -0.79 & 1.27 & 2.27 & 1.24 & \\
\hline Average & -3.02 & -1.0 & - 0.5 & 0.05 & \\
\hline \multirow[t]{2}{*}{8} & -2.21 & 3.04--3.21 & -3.08--3.14 & -3.35 & \multirow[t]{2}{*}{Thermometer Temp. Sensors} \\
\hline & -0.50 & -1.10--1.70 & -1.68--2.20 & -2.78 & \\
\hline Average & -1.35 & -2.15 & -2.41 & -3.1 & \\
\hline \multirow[t]{2}{*}{Setup\#} & \multicolumn{2}{|l|}{with al. plate w/o} & \multicolumn{3}{|l|}{w/o al. plate (inside the al. cage which obstructs the air flow)} \\
\hline & \multicolumn{2}{|l|}{Ther / T.Sensor Ther} & T.Sensor & \multicolumn{2}{|l|}{Remarks} \\
\hline 2a & \multicolumn{2}{|l|}{6.23/8.89 3.5} & .50 / 6.06 & \multicolumn{2}{|l|}{--} \\
\hline 3 a & \multicolumn{2}{|l|}{\(9.55 / 10.98\) 6 6.12} & . 12 / 5.33 & \multicolumn{2}{|l|}{--} \\
\hline 3b & 10.64 / 1 & & 0.82 / 12.84 & \multicolumn{2}{|l|}{slot in the al. cage at 2 ' height open. This temp. diff. increases by 1 degCel., if we put 3 fans of 100 cfm at open slot.} \\
\hline
\end{tabular}

\section*{V Daily Readings :}

Please refer the file temp_readings.ods attached with this report.

\section*{VI Air Flow meter User Manual :}


\section*{Introduction :}

Congratulations on your purchase of the Extech HD350. This handheld meter measures and displays air velocity (speed), air flow (volume), ambient air temperature, and gauge/differential pressure. This meter is shipped fully tested and calibrated and, with proper use, will provide years of reliable service.```

